

- 1) In the two-dimensional incompressible case, let ρ , p , and \bar{V} are density, pressure, and velocity. The Navier-Stokes equation is given as

$$(1) \quad \begin{cases} \rho \frac{D\bar{V}}{Dt} = \rho f - \text{grad } p + \mu \nabla^2 \bar{V} \\ \text{div } \bar{V} = 0 \end{cases}$$

where $\frac{D}{Dt}$ is material derivative and ∇^2 is the Laplace operator.

- (a)(6%) If $\rho = \text{constant}$ and $f = \text{grad } \phi$, prove that (1) implies the vorticity equation

$$\frac{Dw}{Dt} = (w \cdot \text{grad})\bar{V} + \frac{\mu}{\rho} \nabla^2 w$$

where $w = \text{curl } \bar{V}$

- (b) (7%) If ℓ , ν/ℓ^2 , U , and $\rho_0 \nu U/\ell$ where $\nu = \mu/\rho_0$ are length, time, velocity, and pressure scales, show that in dimensionless form the governing equation (1) can be written

$$\text{div } \bar{V} = 0, \quad \bar{V}_t + \text{Re}(\bar{V} \cdot \text{grad})\bar{V} = -\text{grad } p + \nabla^2 \bar{V}$$

where $\text{Re} = U\ell/\nu$.

- (c) (7%) For flows with small Reynolds number, that $\text{Re} \ll 1$, show that

$$\nabla^2 p = 0, \quad \frac{\partial}{\partial t}(\nabla^2 \bar{V}) = \nabla^2(\nabla^2 \bar{V}).$$

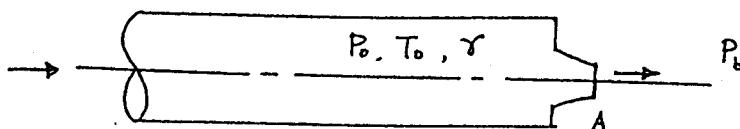
- 2.) A flow of stagnation pressure P_0 and stagnation temperature T_0 is discharging through a converging nozzle of exit area A to a back pressure of P_b . Assume steady flow of perfect gas and specific heat ratio as γ .

- (a) Derive the mass flow rate \dot{m} at nozzle exit as function of P_0 , T_0 , P_b , A and γ .

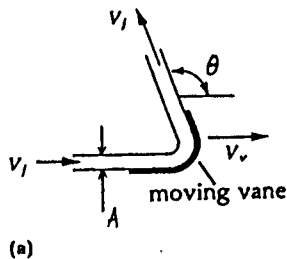
(10%)

- (b) Prove that the mass flow rate is maximum when the nozzle is choking.

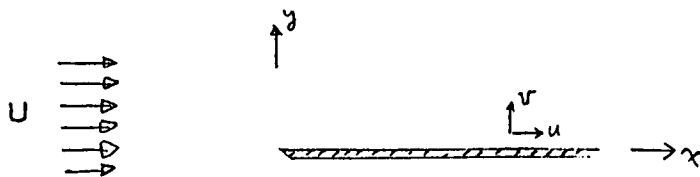
(10%)



- 3). (a) A jet of water with density ρ striking a curved vane, which turns the jet through an angle of θ , as shown in Fig. a. The vane is moving to the right at a velocity of V_v , and the jet has a cross-sectional area A . Develop an expression for forces, F_x and F_y , exerted on the vane, assuming no frictional loss. (15%)
- (b) When the vane is stationary, determine F_x and F_y . (5%)



- 4). a. 下圖為在無限大空間之均勻流場中的一塊平板。試繪圖說明在平板上方的流場內，那一些區域是流體黏性會發生作用的區域，為什麼？(3%)



- b. 說明什麼情形下，流體黏性發生作用區域會變成邊界層。(1%)
- c. 什麼情形下，邊界層內的流體會由層流變成紊流。並繪圖說明邊界層內，層流區及紊流區的速度分佈示意圖，並說明層流與紊流層之速度有何不同，為什麼不同。(8%)
- d. 設 $\frac{\partial p}{\partial x} = 0$ ，試由下面的 Navier Stokes 方程式，使用數量級大小比較法，證明邊界層中， $\frac{\partial p}{\partial y} \sim 0$ 。(8%)

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left[\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right]$$

$$u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial y} + \nu \left[\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right]$$

5) The application of the hydrodynamic theory of Couette flow in the study of lubrication can be illustrated best by an example of the slipper bearing (Fig. a). The gap between the slipper block and the bearing guide is always much smaller than the length of the block and is filled with a lubricant, usually oil. In order to make the motion steady, a system of coordinates is chosen in which the slipper block is stationary and the bearing guide moves with a velocity $-U$ (Fig. b). For a steady, two-dimensional flow and a small

inclination angle α , we have $v \ll u$, $\frac{\partial}{\partial t} = 0$. u is the velocity component in the x direction and v is the velocity component in the y direction. Please answer the following problems: 20%

- a) Find out the governing equations and boundary conditions. (10%)
- b) Find out the mass flow rate. (10%)

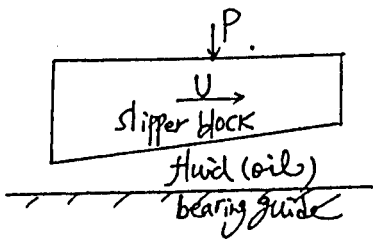


Fig. a

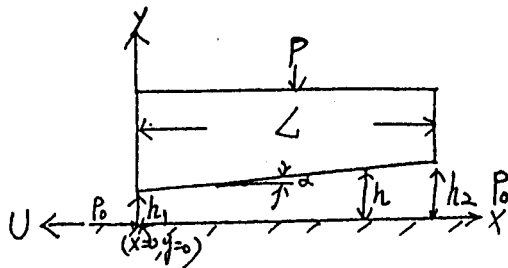


Fig. b

$$\left(\frac{h}{L} \approx 10^{-3}, Re = \frac{\rho U L}{\mu} < 2.5 \times 10^4 \right)$$