

1. A fluid with density ρ , viscosity μ , and velocity V flows through a pipe with diameter D and length L . The frictional pressure loss Δp in the pipe flow can be expressed by other variables:

$$\Delta p = \Delta p(\rho, V, \mu, D, L, a) \quad (1)$$

where a is the speed of sound. You are asked to perform a dimensional analysis for Eq. (1) using the dimensional system, mass-length-time, and assuming

$$\Delta p = C_1 \rho^{b_1} V^{b_2} \mu^{b_3} D^{b_4} L^{b_5}, \quad C_1 = \text{const.}$$

in which $b_i, i = 1, 2, \dots, 5$, are to be determined. Note that the equation you obtained should be expressed in a dimensionless form. (15%) Moreover, identify some common parameters (their common names) (5%).

2. Consider an air flow in a circular pipe, see the figure. Flow passes the screen at $x = 0$ which results into momentum loss.

Assume: (i) At $x = -1d$, flow is uniform in velocity distribution with the velocity U_1 and the pressure P_1 measured.

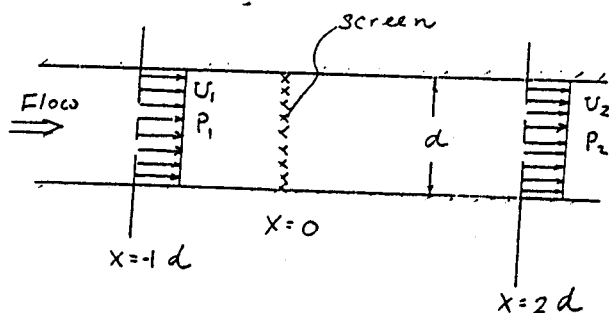
(ii) At $x = 2d$, flow is uniform in velocity distribution with the velocity U_2 and the pressure P_2 measured.

Questions: (20%)

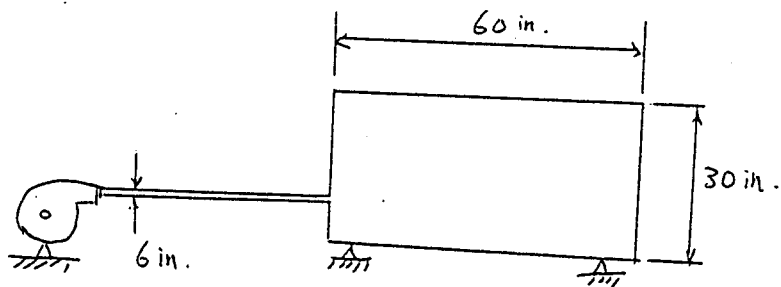
(i) Is $U_1 = U_2, U_1 > U_2,$ or $U_1 < U_2?$ why?

(ii) Is $P_1 = P_2, P_1 > P_2,$ or $P_1 < P_2?$ why?

(iii) Can you apply the Bernoulli's equation for flow in the entire region $-1d < x < 2d?$



3. An air compressor is used to pressurize an initially evacuated tank. The tank is 30 inches in diameter and 60 inches long. The supply line is 6 inches in diameter and conveys a flow of 5 ft/s. The air compressor's output pressure and temperature are constant at 50 psia and 90° F. The tank temperature of 70° F is also constant. Calculate the time required for the tank pressure to reach 15 psia. (20%)



4. For the steady incompressible flat-plate laminar boundary-layer flow, the velocity profile is assumed as follows:

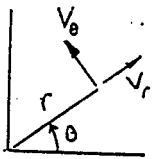
$$\frac{u}{U} = \frac{3}{2}\left(\frac{y}{\delta}\right) - \frac{1}{2}\left(\frac{y}{\delta}\right)^3$$

By using the integral momentum analysis, compute (a) $\left(\frac{\theta}{x}\right)\sqrt{Re_x}$, (b)

$\left(\frac{\delta^*}{x}\right)\sqrt{Re_x}$, (c) $\left(\frac{\delta}{x}\right)\sqrt{Re_x}$, (d) $C_f\sqrt{Re_x}$ and (e) $C_D\sqrt{Re_x}$ (20%)

θ : momentum thickness, δ^* : displacement thickness, δ : boundary-layer thickness
 C_f : friction coefficient, C_D : drag coefficient, Re_x : Reynolds number
 U : freestream velocity

5. Consider a steady and inviscid flow with negligible gravity. Answer the following questions: (20%)



(polar coordinates)

- a) The stream function of a uniform stream (freestream velocity = U) passing a solid body is expressed as

$$\Psi = Ur\left(1 - \frac{a^2}{r^2}\right) \sin \theta, \quad a = \text{constant}$$

- What would be the shape of this solid body if $\Psi = 0$ is chosen to be the surface stream function?
- b) What would the flowfield be modified if an additional term is added to the previous stream function to result in a new form

$$\Psi = Ur\left(1 - \frac{a^2}{r^2}\right) \sin \theta + \frac{G}{2\pi} \ln r, \quad G < 4\pi aU$$

Does the region of the flowfield $r > a$ become rotational after this modification? State your reasons about why you made such an assertion.

- c) The radial and circumferential velocity components can be expressed as

$$V_r = \frac{1}{r} \frac{\partial \Psi}{\partial \theta}$$

$$V_\theta = -\frac{\partial \Psi}{\partial r}$$

Determine V_r and V_θ at $r = 2a$ and find the line integral around a circle of $r = 2a$

$$\oint_{r=2a} r V_\theta d\theta = ?$$

Is there any physical meaning associated with this line integral?

- d) Determine the lift force exerting on this body as the stream function in question b) is taken to represent the flowfield.