國立成功大學

114學年度碩士班招生考試試題

編 號: 114

系 所: 生物醫學工程學系

科 目:流體力學

日期:0210

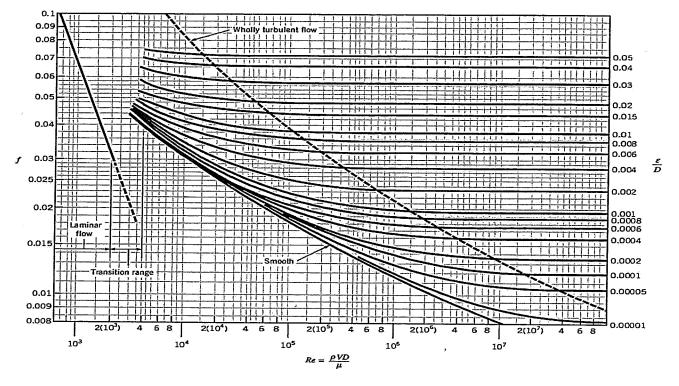
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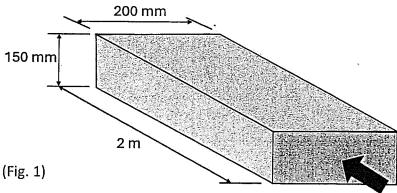
注 意: 1.可使用計算機

2.請於答案卷(卡)作答,於 試題上作答,不予計分。

- 1. Air flows through the galvanized steel duct with a velocity of 4 m/s (Fig.1).
- (a) Determine the <u>hydraulic diameter</u> of the rectangular duct. (5%)
- (b) Determine the flow regime (Hint: turbulent, transition, laminar, or creeping)? (5%)
- (c) Determine the **pressure drop** along a 2-m length of the duct. Take $\rho_a = 1.202 \text{ kg/m}^3$, $v_a = 15.1 \times 10^{-6} \text{ m}^2/\text{s}$. (15%)

| | Smooth surface | Galvanized steel | Riveted steel | Cast iron | Drawn tubing | Concrete |
|----------------|----------------|------------------|---------------|-----------|--------------|----------|
| Roughness (mm) | 0 | 0.15 | 0.9 | 0.26 | 0.015 | 0.3 |

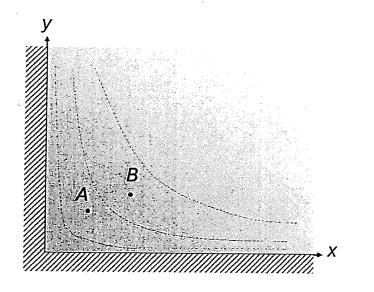




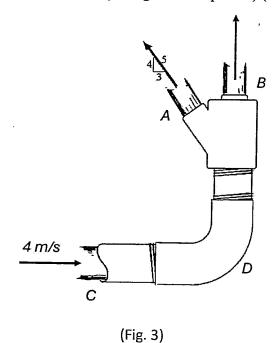
- 2. The stream function for horizontal flow near the corner is defined by $\psi = (8xy)$ m²/s, where x and y are in meters (Fig. 2).
- (a) Show the flow <u>velocities u and v in terms of x and y. (8%)</u>
- (b) Show whether the flow is <u>rotational or irrotational?</u> (5%)
- (c) If the pressure at point A (1 m, 2 m) is 150 kPa, determine the pressure at point B (2 m, 3 m). Take $\rho = 980 \text{ kg/m}^3$. (15%)

(Fig. 2)

(d) Explain why there won't be vortices forming at the sharp corner O in this case even at very high speed flow? (4%)



- 3. Water flows through the pipe C at 4 m/s as shown in Fig. 3. Neglect the size and weight of the pipe and the water within it. The pipe has a diameter of 60 mm at C, and at A and B the diameters are 20 mm.
- (a) Draw a control volume (CV) and define the term CV in fluid mechanics. (3%)
- (b) Determine the pressure at C (Hint: Bernoulli Equation). (8%)
- (c) Determine the <u>horizontal and vertical components of force</u> exerted by elbow *D* necessary to hold the pipe assembly in equilibrium. (Notice: a Free Body Diagram is required!) (12%)



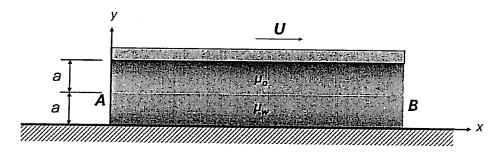
4. The water and oil films have the same thickness a and are subjected to the movement (U) of the top plate in the x-axis (Fig. 4). **Derive velocity profiles for both fluids** in steady state. Noted that there is no pressure gradient between A and B, and no gravity. The viscosities of water and oil are μ_W and μ_O , respectively. (20%)

Navier-Stokes equations for an incompressible Newtonian fluid with dynamic viscosity (μ) rectangular coordinates (x, y, z):

$$\rho \left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} \right) = -\frac{\partial p}{\partial x} + \mu \left[\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} + \frac{\partial^2 u_x}{\partial z^2} \right] + \rho f_x$$

$$\rho \left(\frac{\partial u_y}{\partial t} + u_x \frac{\partial u_y}{\partial x} + u_y \frac{\partial u_y}{\partial y} + u_z \frac{\partial u_y}{\partial z} \right) = -\frac{\partial p}{\partial y} + \mu \left[\frac{\partial^2 u_y}{\partial x^2} + \frac{\partial^2 u_y}{\partial y^2} + \frac{\partial^2 u_y}{\partial z^2} \right] + \rho f_y$$

$$\rho \left(\frac{\partial u_z}{\partial t} + u_x \frac{\partial u_z}{\partial x} + u_y \frac{\partial u_z}{\partial y} + u_z \frac{\partial u_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_z}{\partial y^2} + \frac{\partial^2 u_z}{\partial z^2} \right] + \rho f_z$$



(Fig. 4)